

Turbulent Thermal Transport

D. Biles, A. Ebadi, C.M. White
University of New Hampshire

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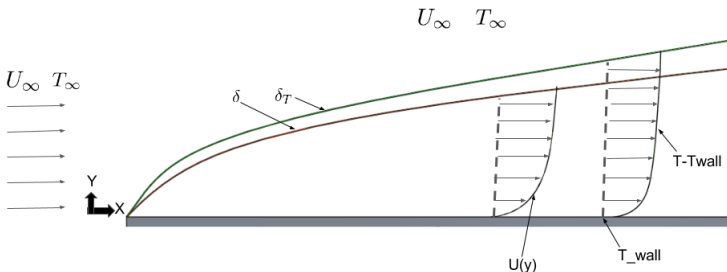
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PARTNERSHIP

AWARD'S: 1258702, 1258594, 1258697



Thermal Boundary Layers 101

-2 field variables to study, Velocity and Temperature



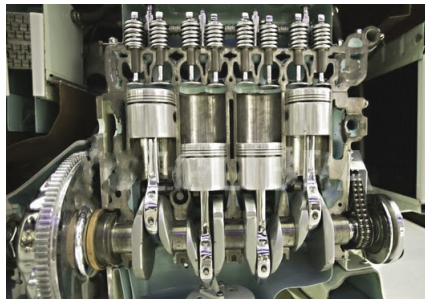
- ▶ U_∞ = Free-stream velocity
- ▶ $U(y)$ = Stream-wise velocity profile
- ▶ δ = Boundary layer thickness
- ▶ $Pr = 0.707 = (\text{viscous diff.}) / (\text{thermal diff.})$
- ▶ T_∞ = Free-stream temperature
- ▶ T_{wall} = Wall temperature
- ▶ δ_T = Thermal boundary layer thickness

Engineering Systems

-Design of engineering systems relies on accurate models of real physics



<http://science.howstuffworks.com>



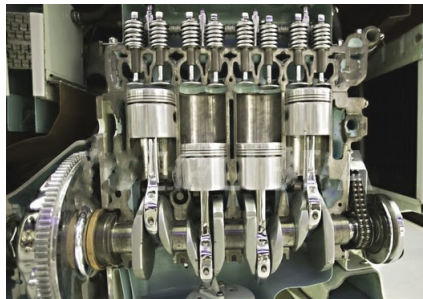
<http://me-mechanicalengineering.com>

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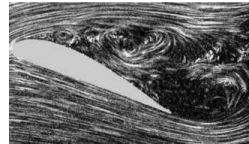
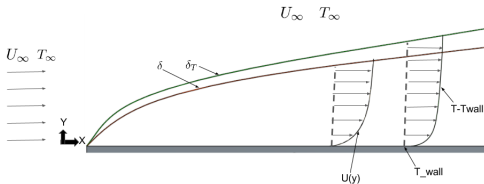
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Important Parameters:

Wall shear stress $\rightarrow \tau_{wall}$

Wall heat flux $\rightarrow q''_{wall}$

Creating High Quality Boundary Layer Datasets



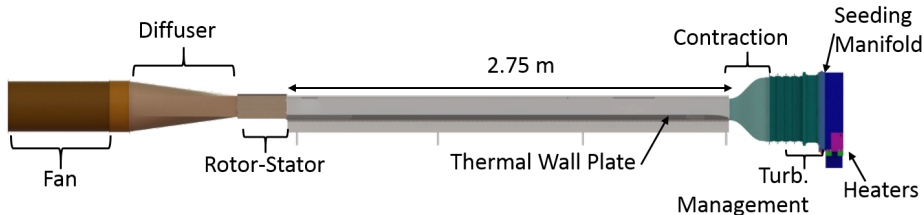
<http://en.wikipedia.org/wiki/flow/separation>

- ▶ Need to create test-bed for developing equilibrium/non-equilibrium flows
- ▶ Require experimental data sets to improve/validate models
- ▶ Requires controlling all boundary conditions

Experimental Facility

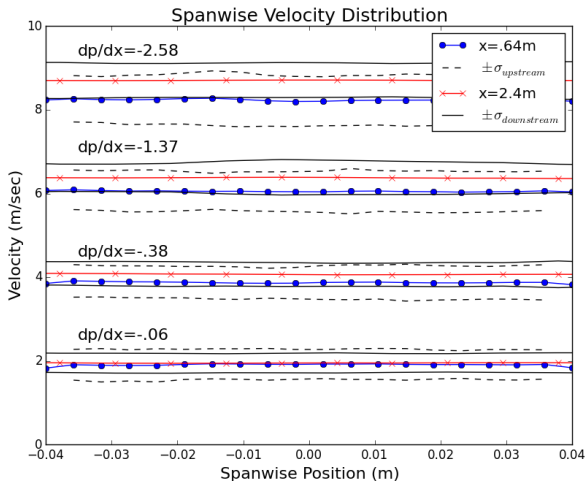
Non-Equilibrium And Thermal boundary layer Tunnel (N.E.A.T.)

- Length=2.75m
 - Turbulent Management section
 - VFD Controlled Motor
 - Bank of Resistive Heaters
 - Thermal Wall Plate
 - Rotor-Stator Mechanism
- | | |
|-------|--|
| ————→ | Development Length |
| ————→ | Free Stream Turbulence |
| ————→ | U_{∞} |
| ————→ | T_{∞} |
| ————→ | T_{wall} |
| ————→ | $\frac{\partial U_{\infty}}{\partial t}$ |

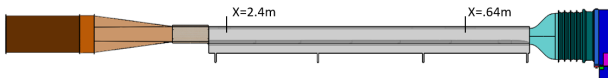


UNH Thermal Boundary Layer Wind Tunnel

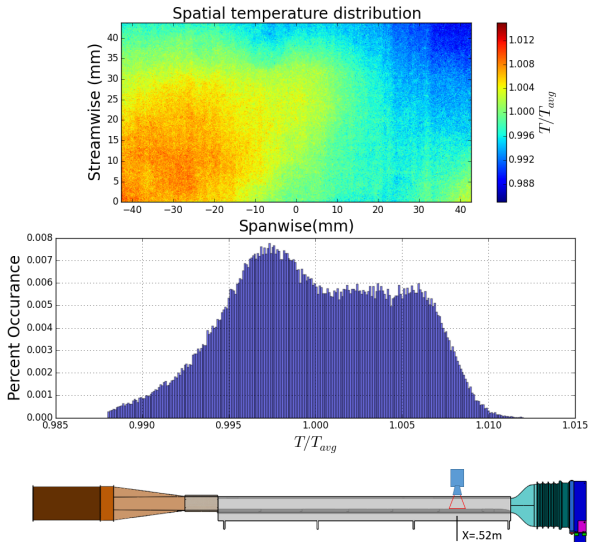
Pressure Gradient



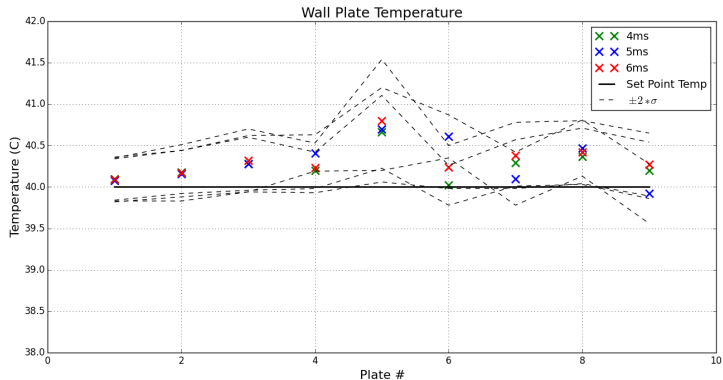
- ▶ Span-wise profile $\pm 12.5\%$
- ▶ dp/dx Parameter $K = \frac{\nu}{U_\infty^2} \frac{dU_\infty}{dx}$
- ▶ $K > 1.25 \times 10^{-6}$
- ▶ at 4m/sec $K = 1.0 \times 10^{-7}$



Spatial Wall Temperature Distribution

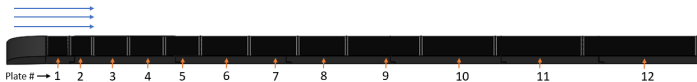


Mean Plate Temperature

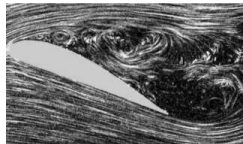
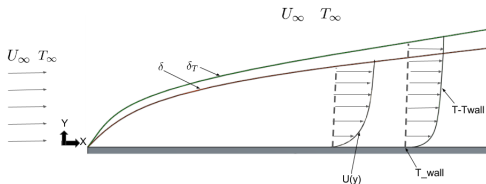


► Set Temperature=40°C

► Controller loop at 20Hz



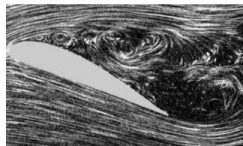
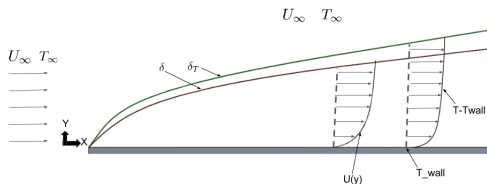
Creating High Quality Boundary Layer Datasets



<http://en.wikipedia.org/wiki/flow/separation>

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Creating High Quality Boundary Layer Datasets

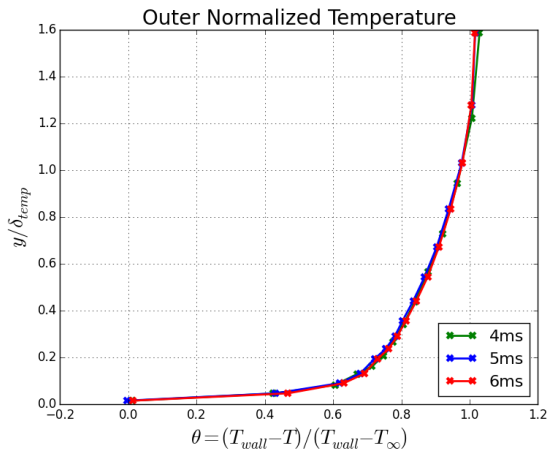


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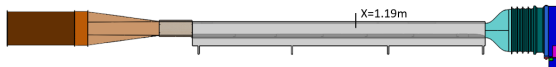
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→ Equilibrium momentum and thermal boundary layer measurements

Temperature Profile



- ▶ Type J thermocouple
- ▶ $\delta_T / \text{Probe Diam} = 33$
- ▶ 30 Points
- ▶ $T_{wall} = 40^{\circ}\text{C}$
- ▶ Sample Freq = 2Hz
- ▶ Sample length=120sec





Governing Equations

Examine Navier Stokes and Energy Equation

$$\frac{d\vec{u}}{dt} + \vec{u}\nabla\vec{u} = \nu\nabla^2\vec{u} + -\frac{1}{\rho}\nabla P$$
$$\frac{dT}{dt} + u\nabla T = Pr\nabla^2 T$$

- ▶ Temporal Change
- ▶ Convection/Advection
- ▶ Diffusion
- ▶ Pressure Gradient

Assumptions:

- ▶ Steady
- ▶ 2D
- ▶ Incompressible

Laminar Analysis

Similarity scaling transforms PDE $\rightarrow$ ODE

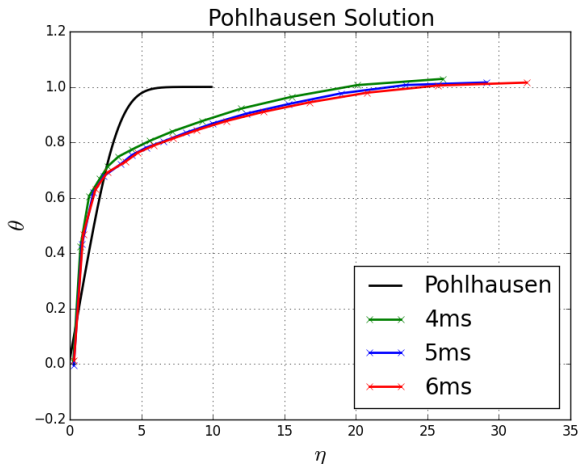
$$\text{Similarity variable: } \eta = y \sqrt{\frac{U_\infty}{\nu x}}$$

$$\text{Blasius Solution (momentum)} \rightarrow 2 \frac{d^3 f}{d\eta^3} + f(\eta) \frac{d^2 f}{d\eta^2} = 0$$

$$\text{Polhausen Solution (energy)} \rightarrow \frac{d^2 \theta}{d\eta^2} + \frac{Pr}{2} f(\eta) \frac{d\theta}{d\eta} = 0$$

$$f' = \frac{U}{U_\infty}, \quad \theta = \frac{T_{\text{wall}} - T}{T_{\text{wall}} - T_\infty}$$

Temperature Profile - Pohlhausen



- Temperature profiles deviate from laminar solution

Laminar Analysis

Similarity scaling transforms PDE $\rightarrow$ ODE

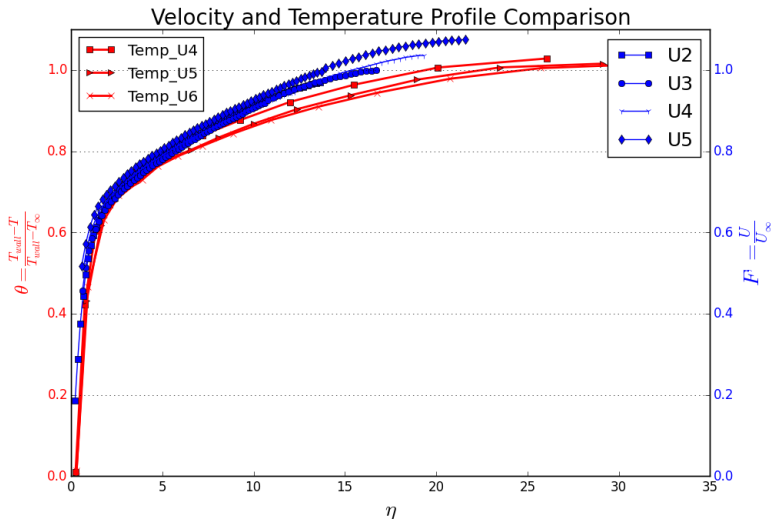
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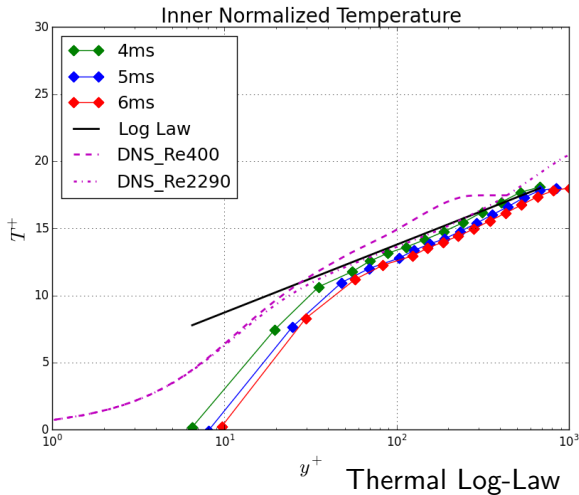
$$f' = \frac{U}{U_\infty}, \quad \theta = \frac{T_{\text{wall}} - T}{T_{\text{wall}} - T_\infty}$$

Temperature Profile - Velocity Profile



Thermal Profile - DNS

DNS (Araya et al.)



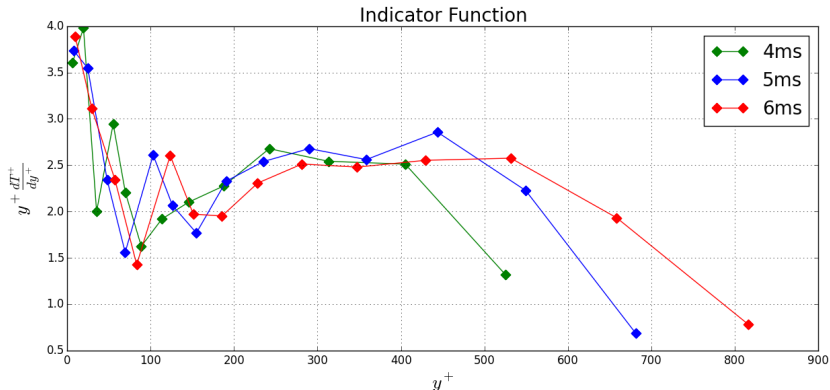
- ▶ $R_{\theta 1} = 400$
- ▶ $R_{\theta 2} = 2290$
- ▶ Constant wall temperature BC
- ▶ $Pr = .71$

Experiment

- ▶ $T^+ = \theta \frac{U_{\infty}}{u_{\tau}} Pr^{2/3}$
- ▶ $y^+ = \frac{\gamma u_{\tau}}{\nu}$
- ▶ Probe Diam = $12.5y^+$

▶ $T^+ = 2.195 \ln(y^+) + 13.2Pr - 5.66$

Thermal Profile - DNS



Thermal Log-Law

- ▶ $T^+ = 2.195 \ln(y^+) + 13.2Pr - 5.66$
- ▶ differentiate both side $\rightarrow y^+ \frac{dT^+}{dy^+} = \text{constant}$



Conclusions/Capabilities

- ▶ Maintain T_{wall} in equil/non-equil flow
- ▶ Pressure gradient is too weak to change mean dynamics
- ▶ Turbulent thermal profiles are observed

Future Work

- ▶ Diagnose TC noise issues
- ▶ Thermocouple measurements in non-equilibrium flow
- ▶ Coupled temperature and velocity measurements in equilibrium environments



Questions??

References

- ▶ 1. Philippe R. Spalart, Jonathan H. Watmuff (1992). "Experimental and Numerical study of a turbulent boundary layer with pressure gradients". J. Fluid Mech. 337-371.
- ▶ 2. Marusic I, Monty JP, Hultmark M, Smits AJ (2013). "On the logarithmic region in wall turbulence". J Fluid Mech 716:R3
- ▶ 3. Kays William Morrow, Crawford Michael E. (1980). "Convective Heat and Mass Transfer". McGraw-Hill Series in Mechanical Engineering. 210-211.
- ▶ 4. Araya Guillermo, Castillo Luciano (2012). "DNS of turbulent thermal boundary layers up to $Re_\theta = 2300$ ". Int Journal of Heat and Mass Transfer. 4003- 4019.



IR Plot

